



Microcooling Developments at University of Twente

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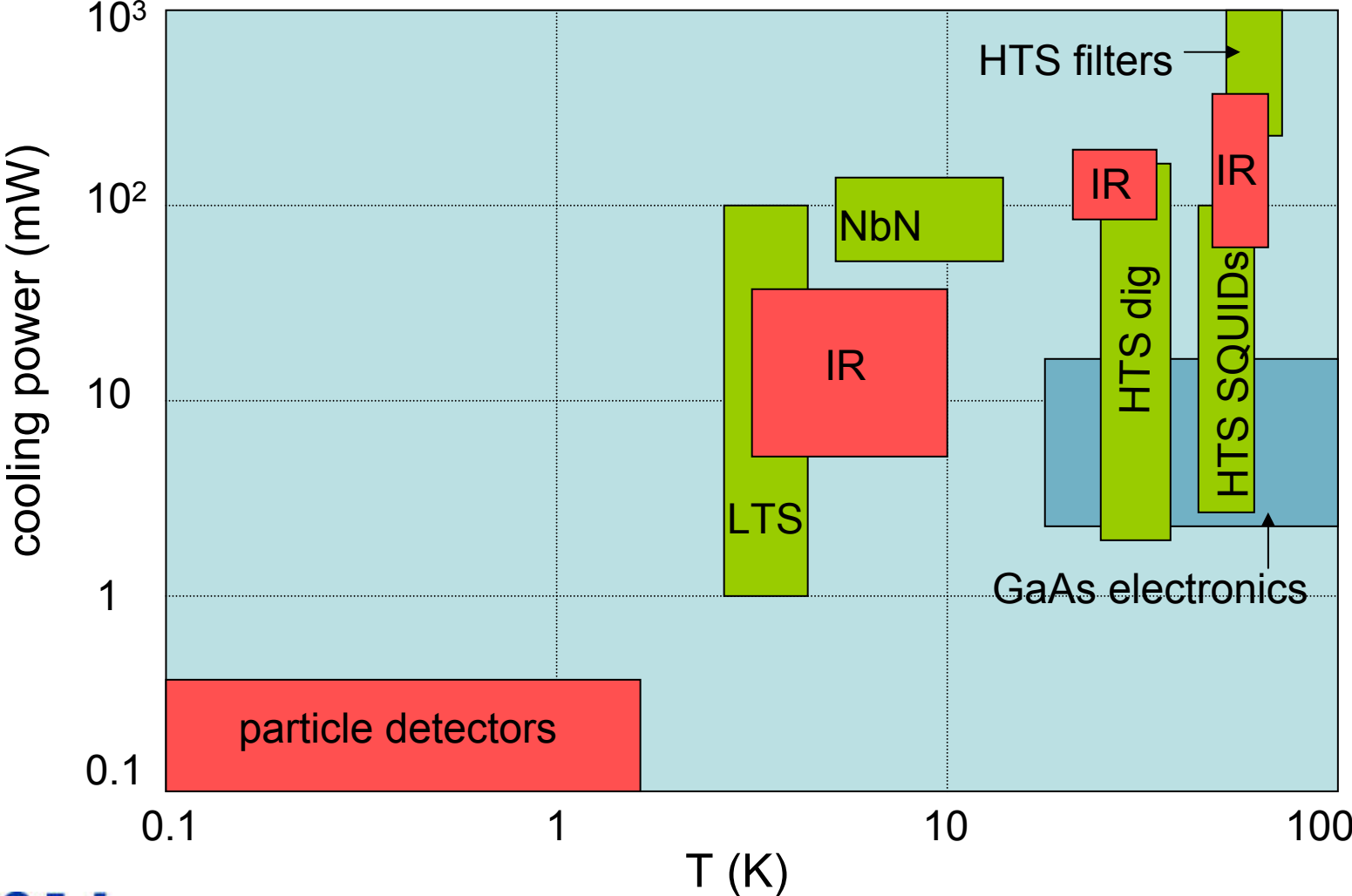
University of Twente,

The Netherlands



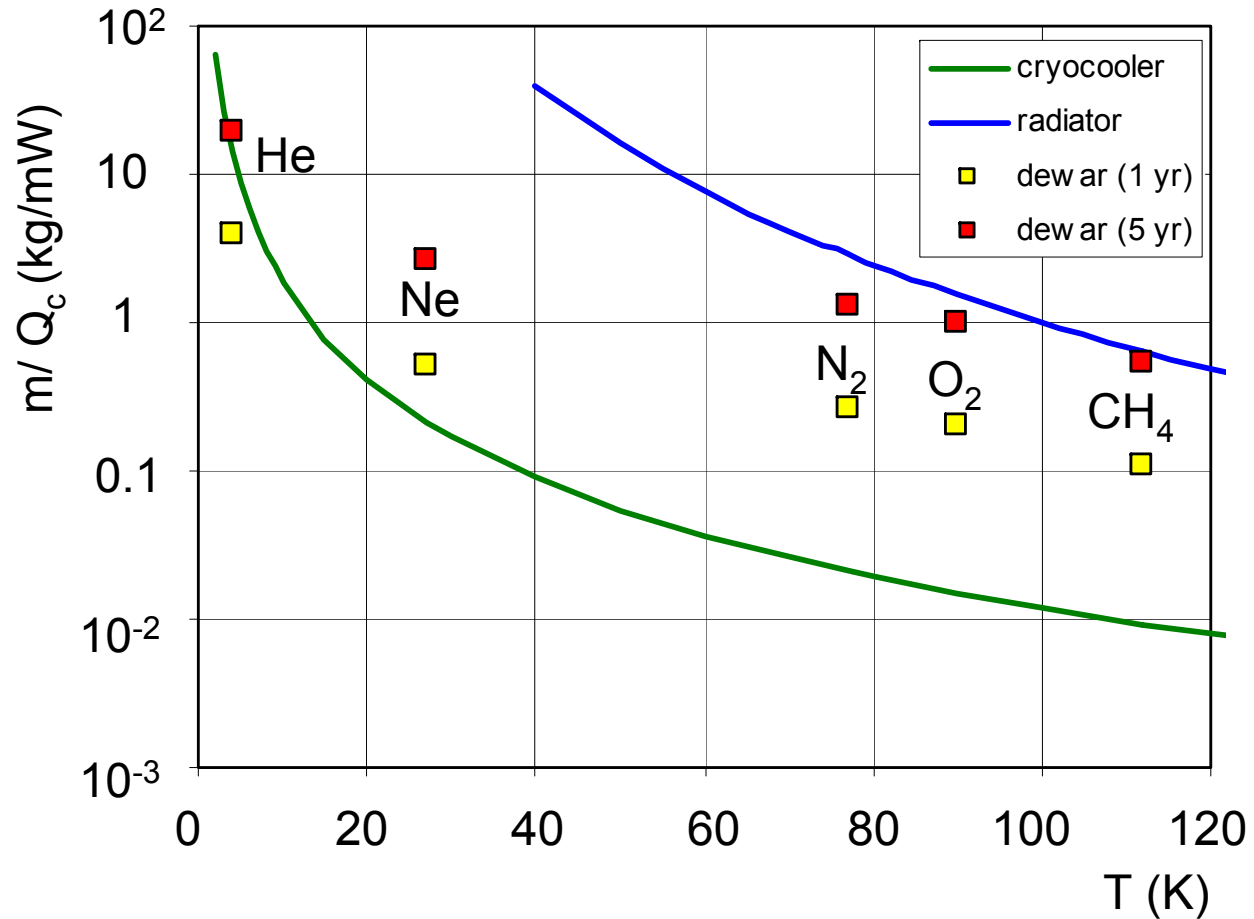
Low temperature division

Cooling needs in Space



Cooling options in Space

mass per unit of cooling power (kg/mW)

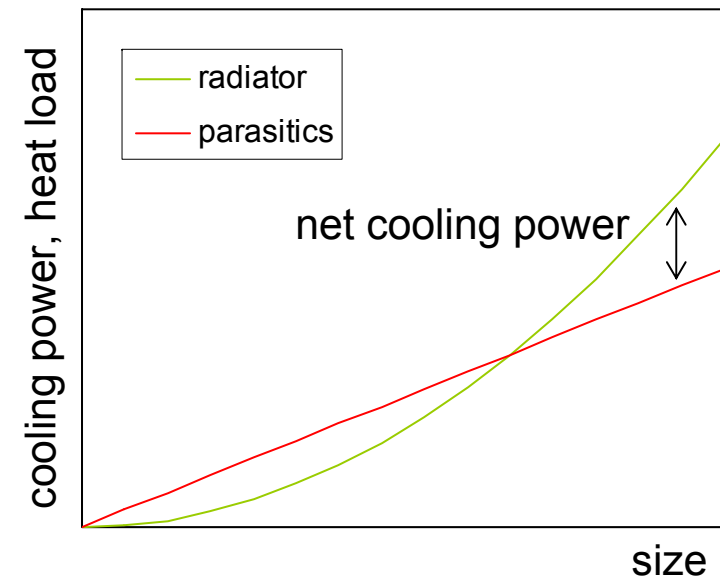
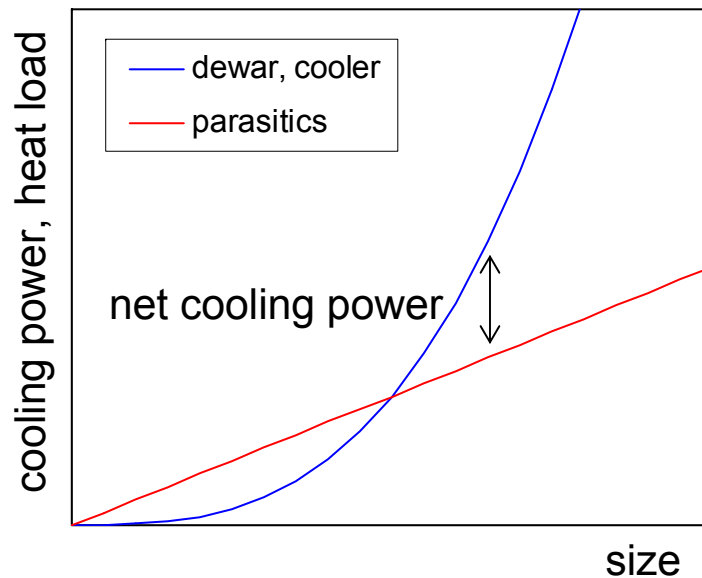


→ lower T means larger cooling system

→ cryocoolers attractive because of size and weight

Scaling

dewar cooling power:	scales with volume (size ³)
cryocooler cooling power:	scales with volume (size ³)
radiator cooling power:	scales with area (size ²)
parasitic heat load:	scales with area/length (size ¹)

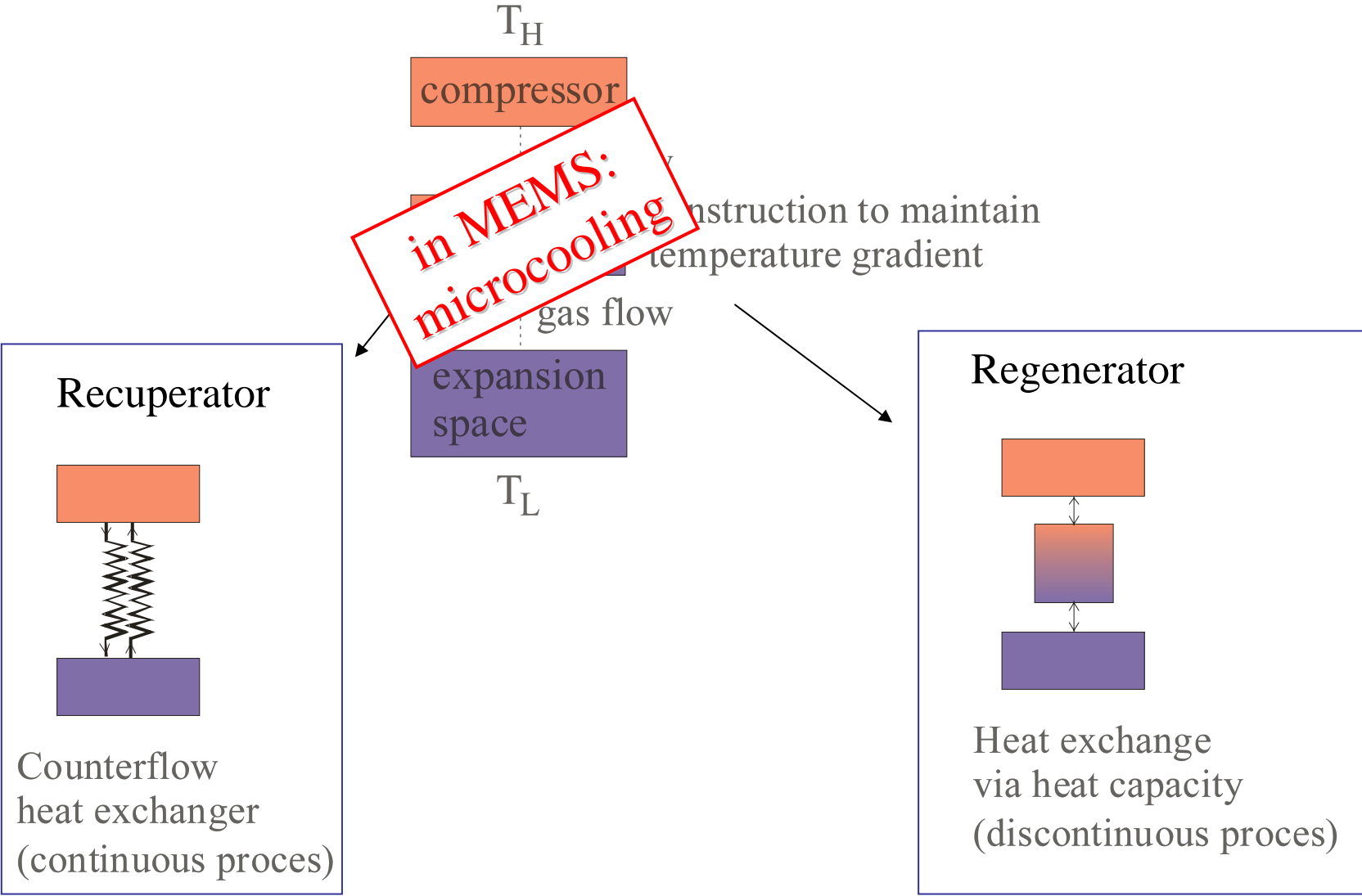


In cryocooler design the dependence of cooling power on size can be influenced

regenerative cooler: $\dot{Q}_L \sim f \times \Delta p \times V_{gas}$ → frequency f can be increased

recuperative cooler: $\dot{Q}_L = \dot{m} \times \Delta H_{gas}$ → compressor design can be used with size-independent flow (e.g. sorption)

Cooling via expansion of compressed gas



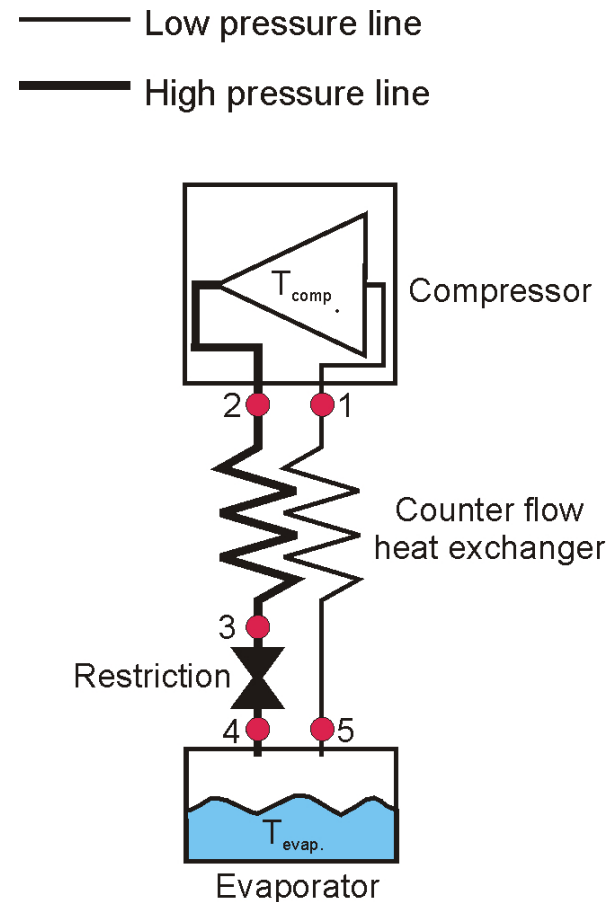
Contents:

- Microcooling research in Twente
 - micro-recuperative cooler
 - micro-regenerative cooler

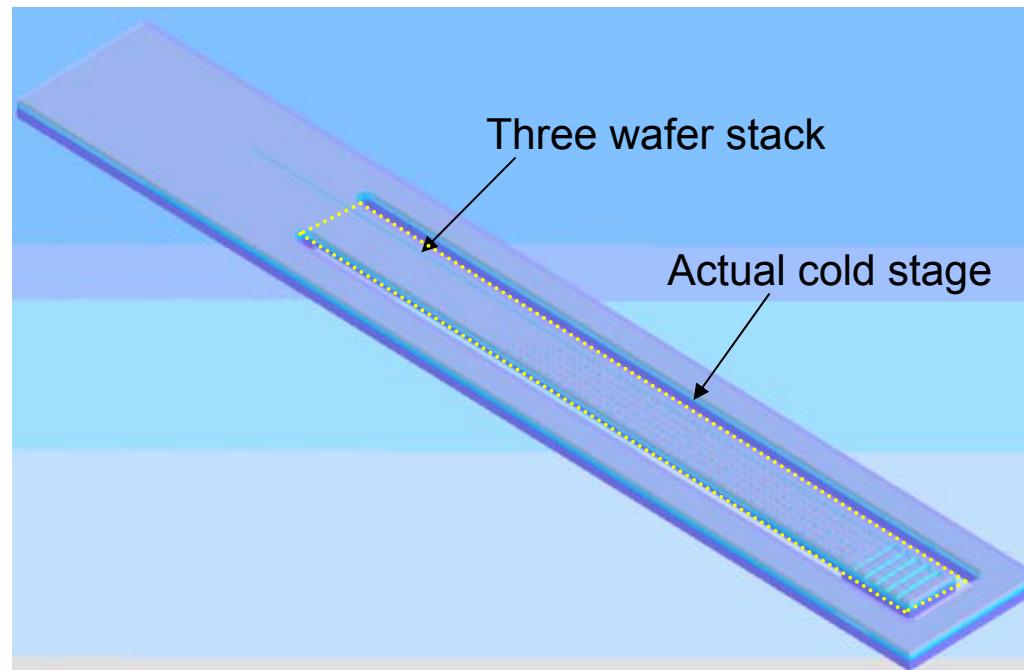
Recuperative Cooler

JT Cycle:

- Gas: N_2
- $P_{\text{high}} = 80 \text{ bar}$
- $P_{\text{low}} = 6 \text{ bar}$
- $T_{\text{cold stage}} = 96 \text{ K}$
- $m_{\text{flow}} = 1 \text{ mg/s}$
- $P_{\text{gross}} \approx 14 \text{ mW}$
- $P_{\text{net}} \approx 10 \text{ mW}$



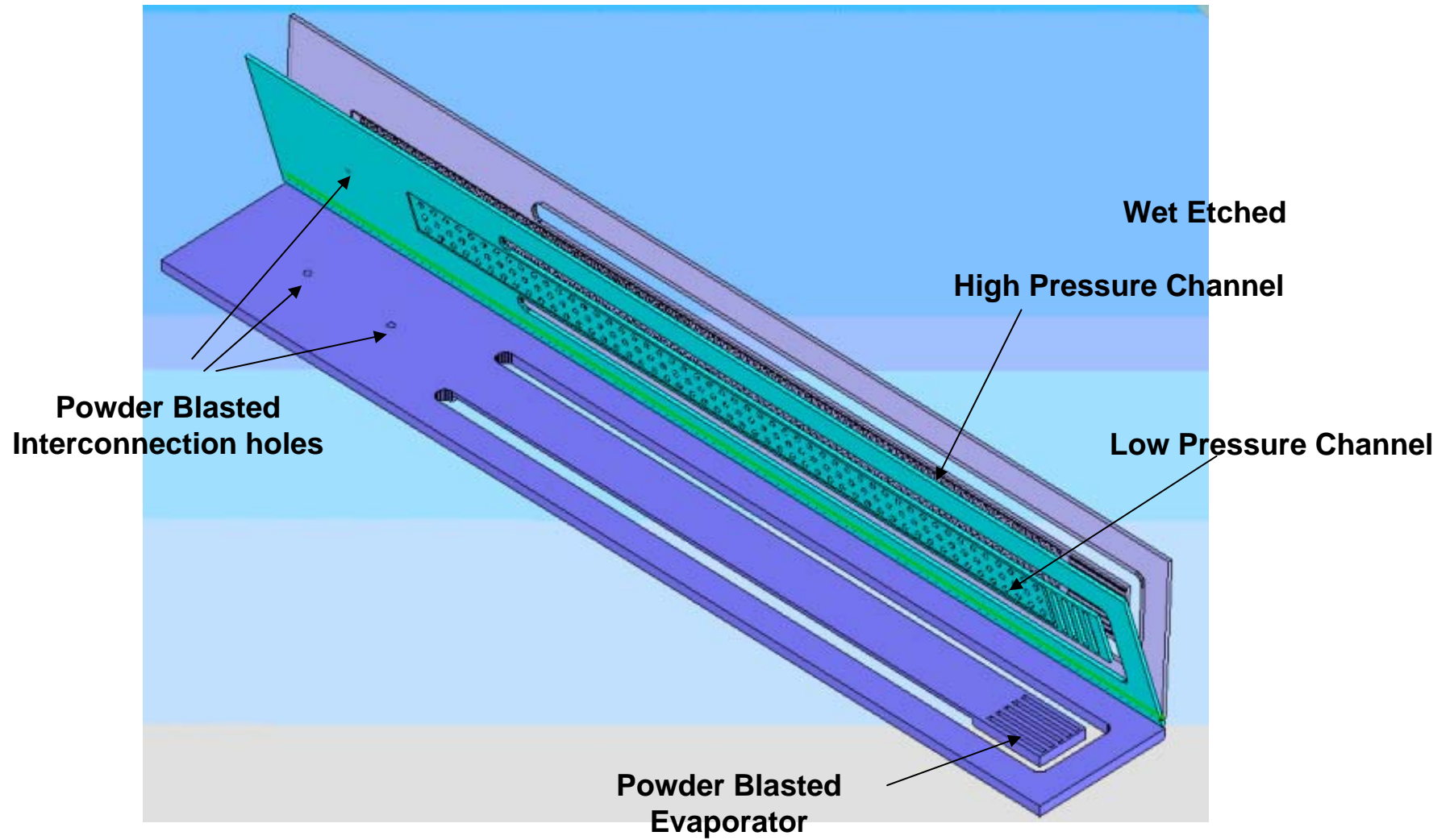
Cold-tip prototype



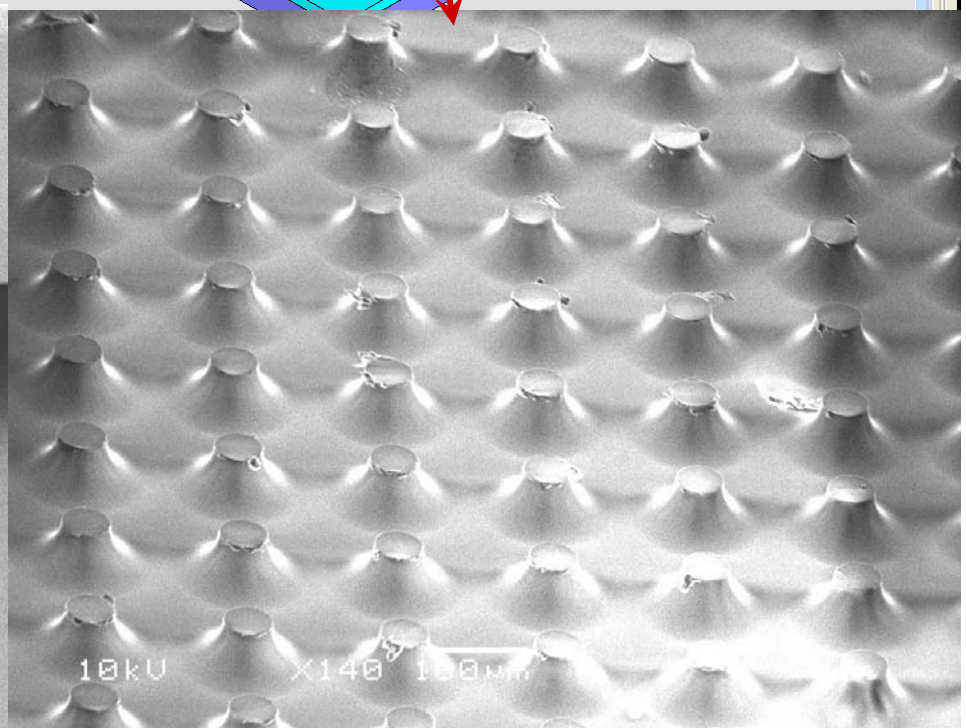
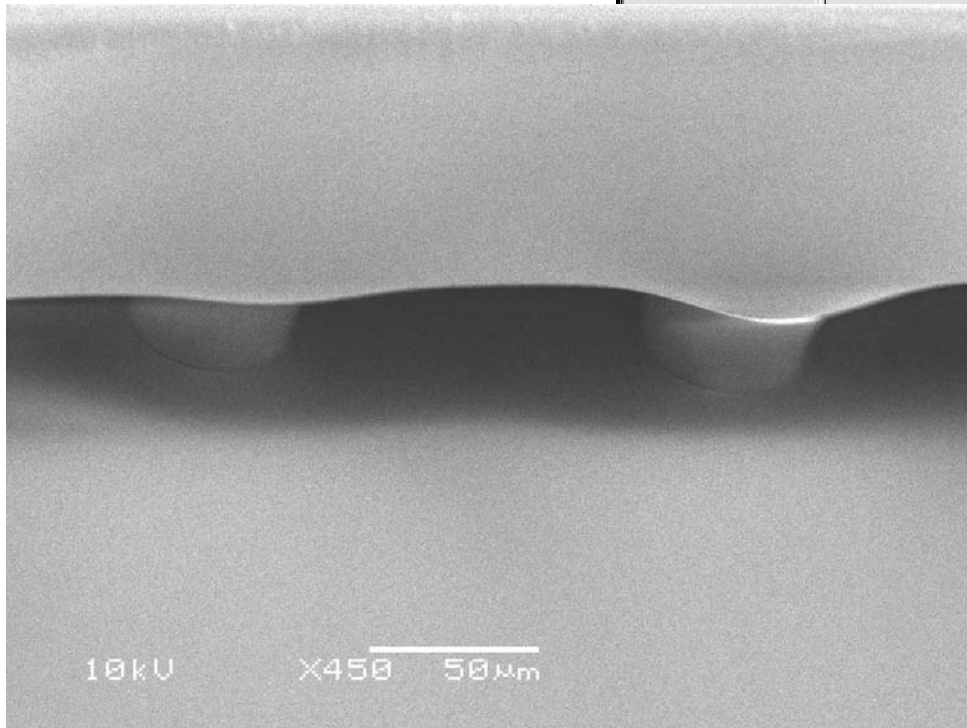
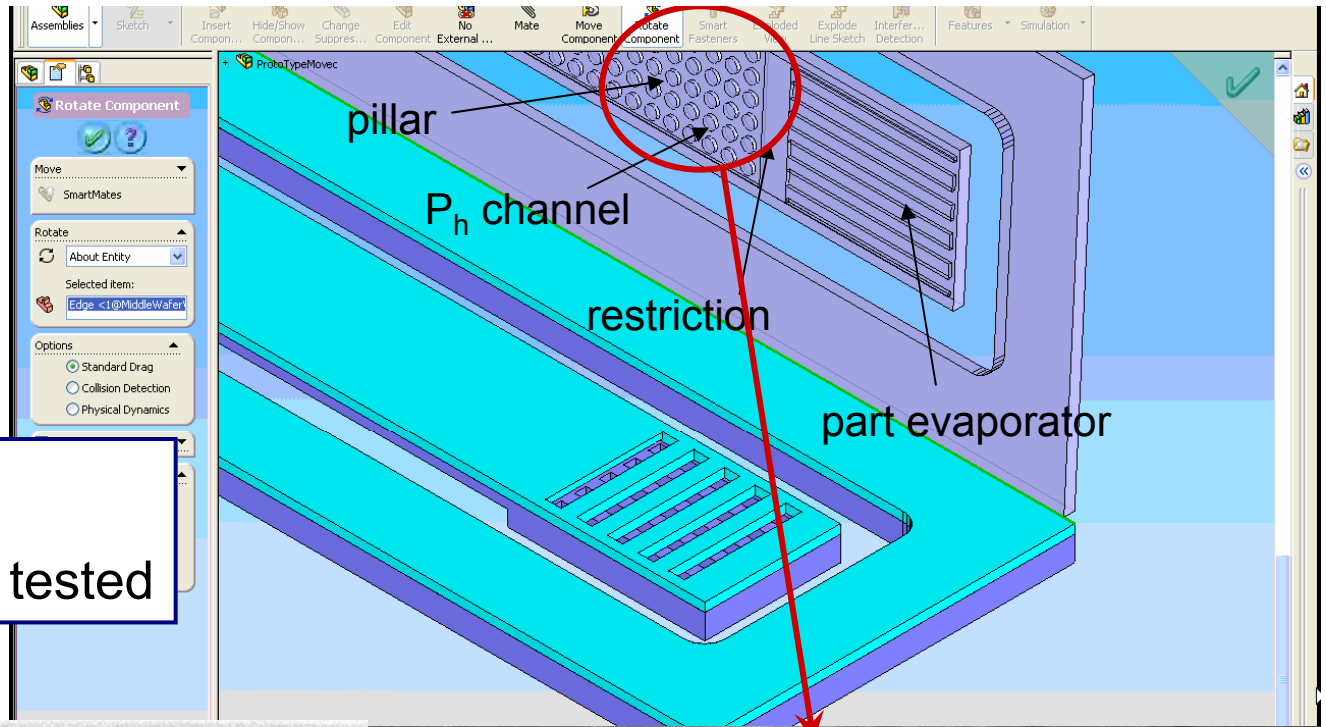
cold-stage length: 28 mm

cold-stage width: 2.2 mm

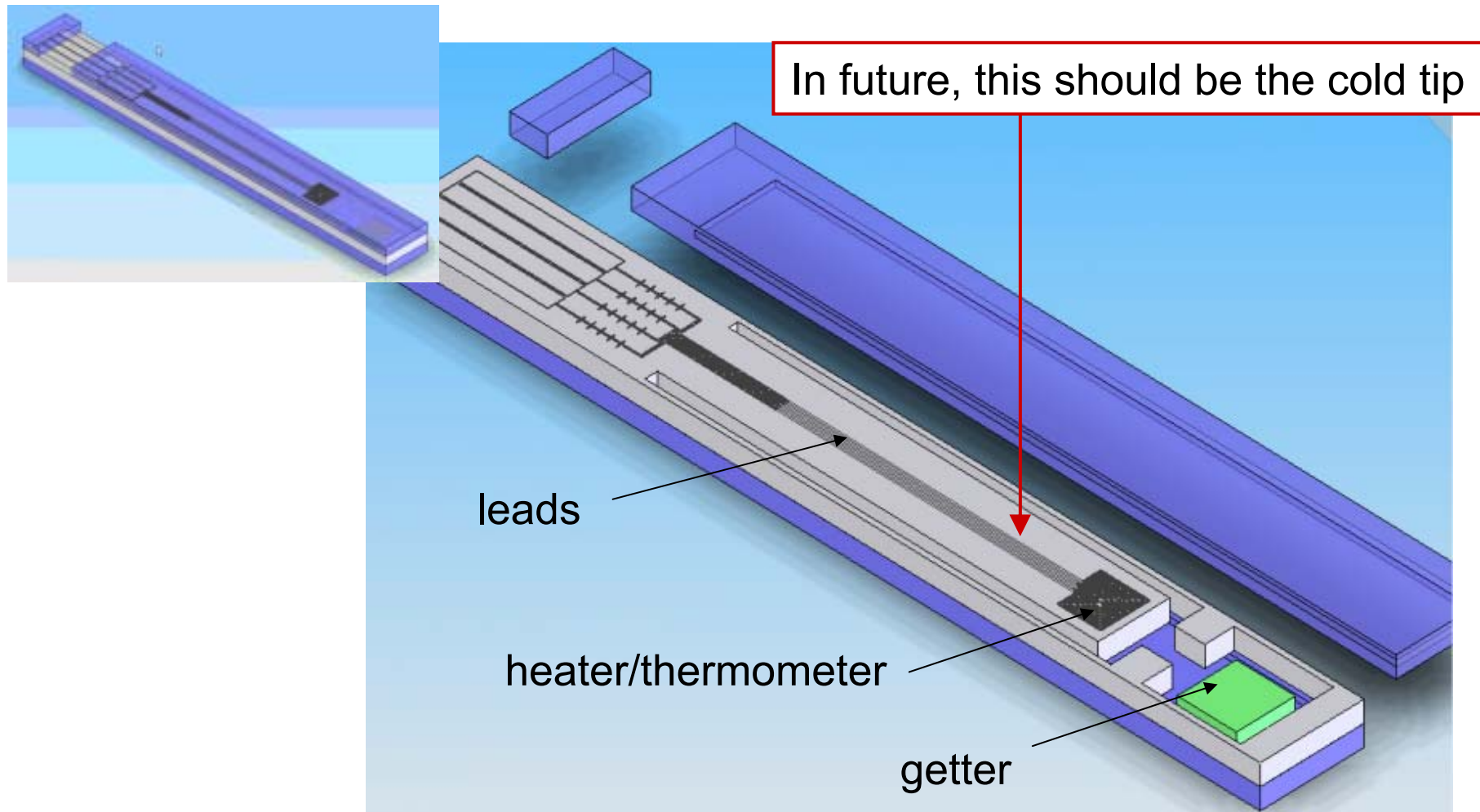
Cold-tip prototype



Design completed,
Prototypes ready to be tested

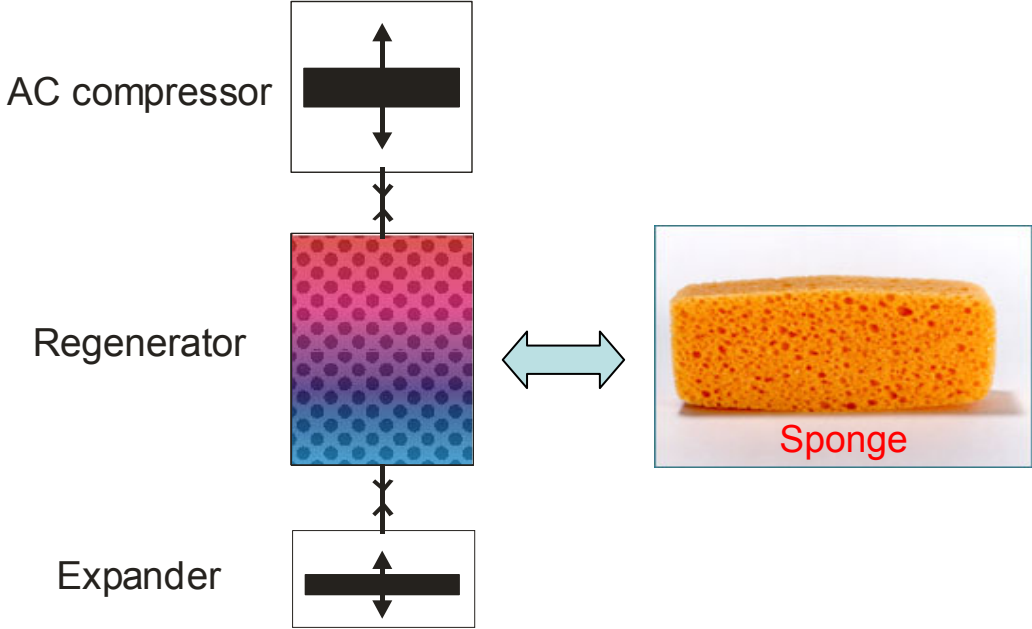
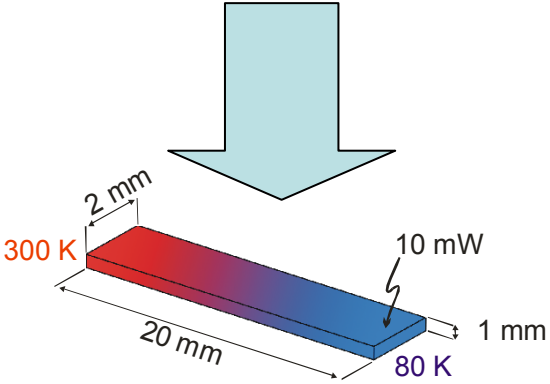


Micro vacuum chamber



→ test chamber under construction: gas pressure to be monitored via thermal conductivity (Pirani-principle)

Micro-regenerative cooling:



Regenerator exchanges heat alternately with gas

Regenerator is critical element

Example:

Cooling Power 30 mW @ 100K

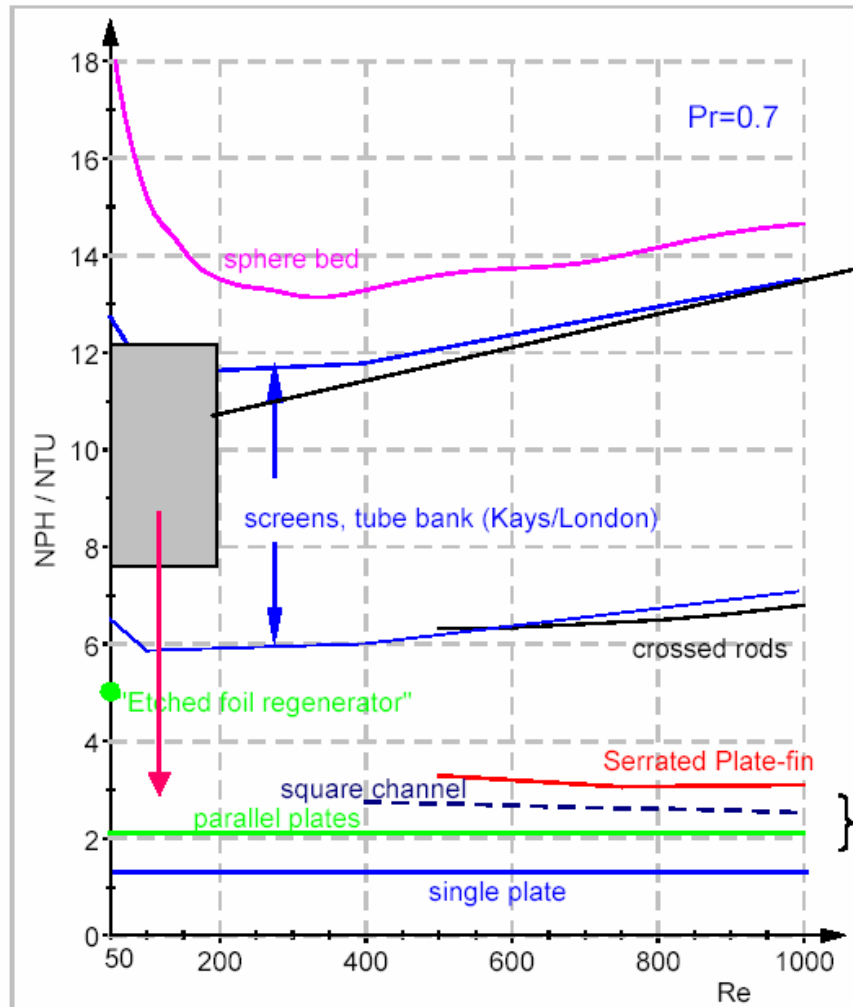
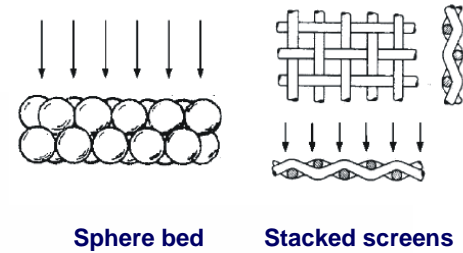
Stored Heat flux in Regenerator

$$\dot{Q}_{reg} = \dot{m} C_{p,g} (T_{amb} - T_{cold}) \approx 400 \text{ mW}$$

If we accept **loss** of **2%**, **→8 mW** lost from cooling power

Optimization of Regenerator is critical!!

MEMS enables regenerator matrix optimization



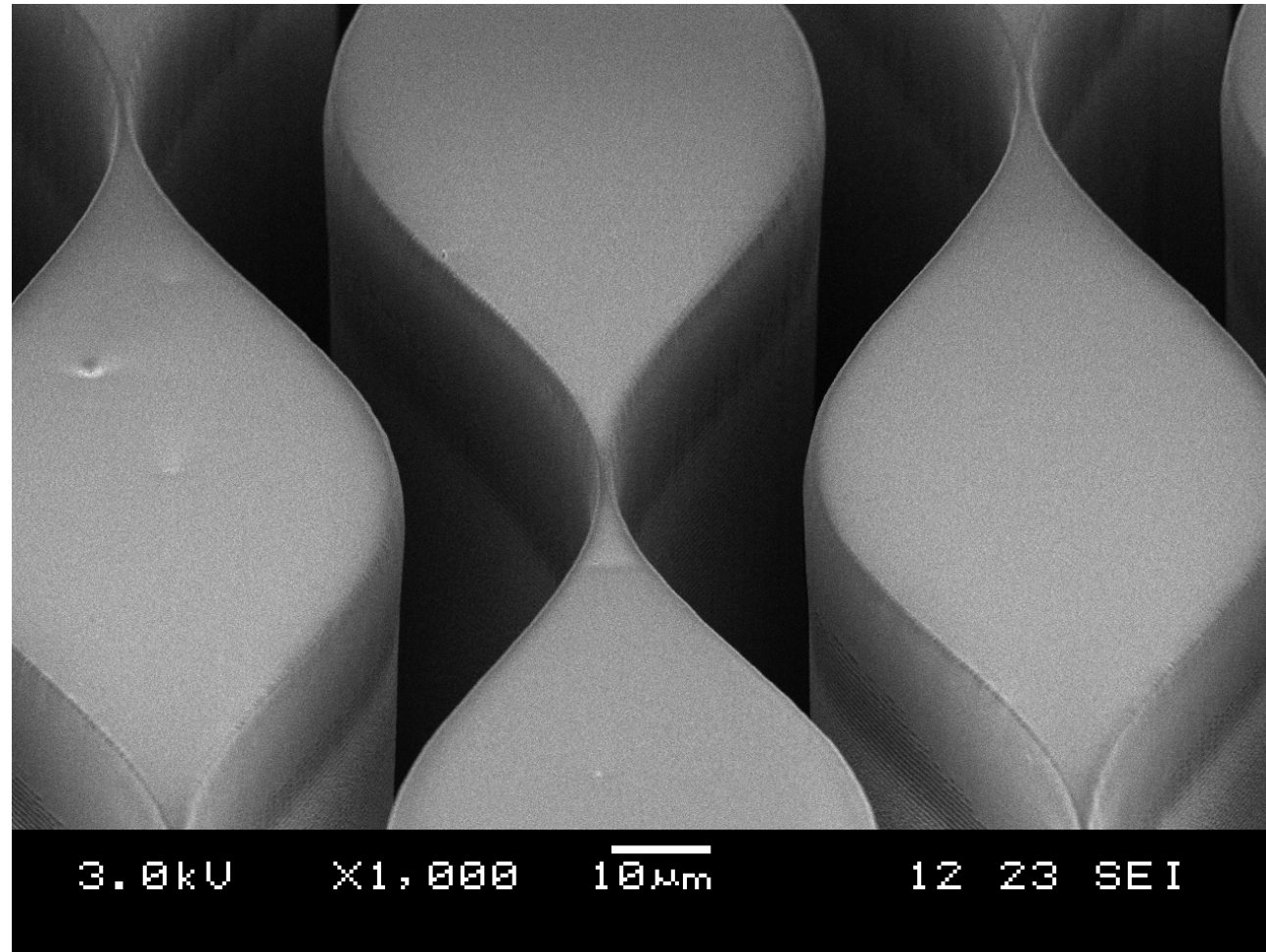
Working area of stacked screens in cryocoolers

Potential for improvement: **factor 4**

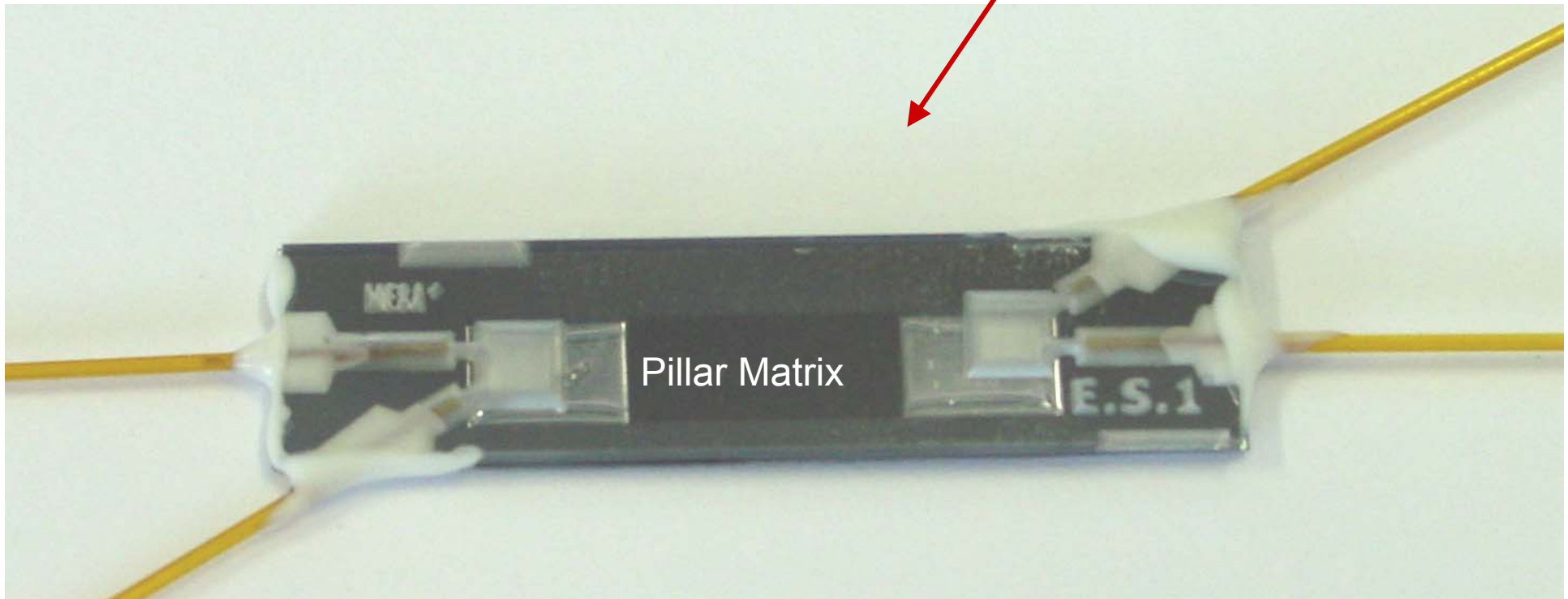
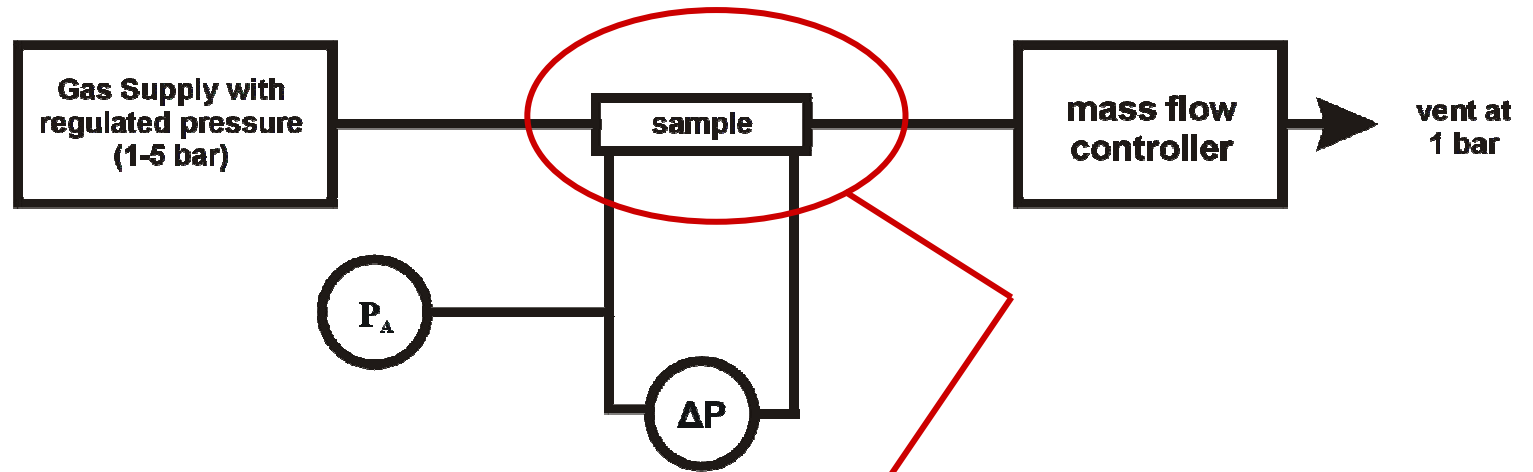
Analytical solutions

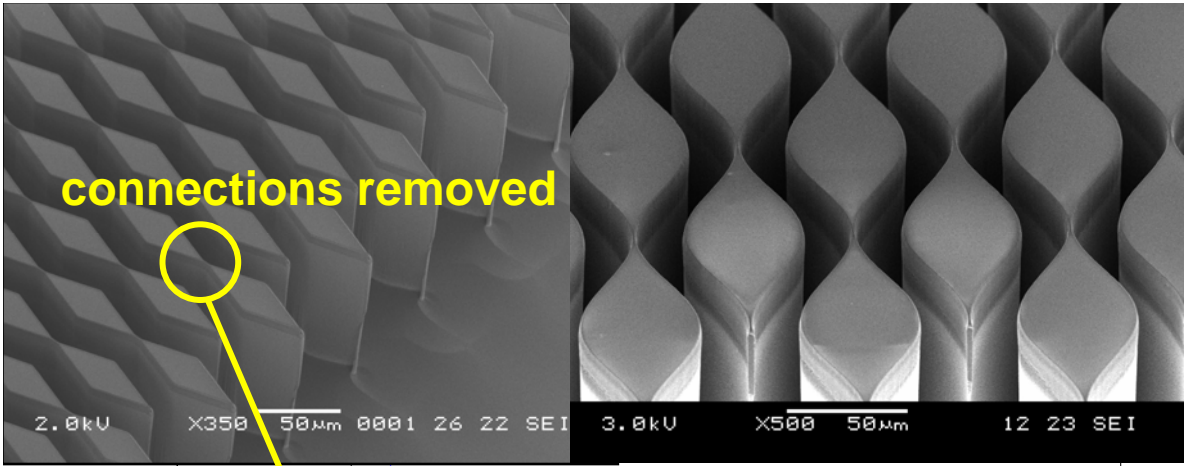
Fabrication

Regenerator Pillar Matrix – Optimized Bosch Process



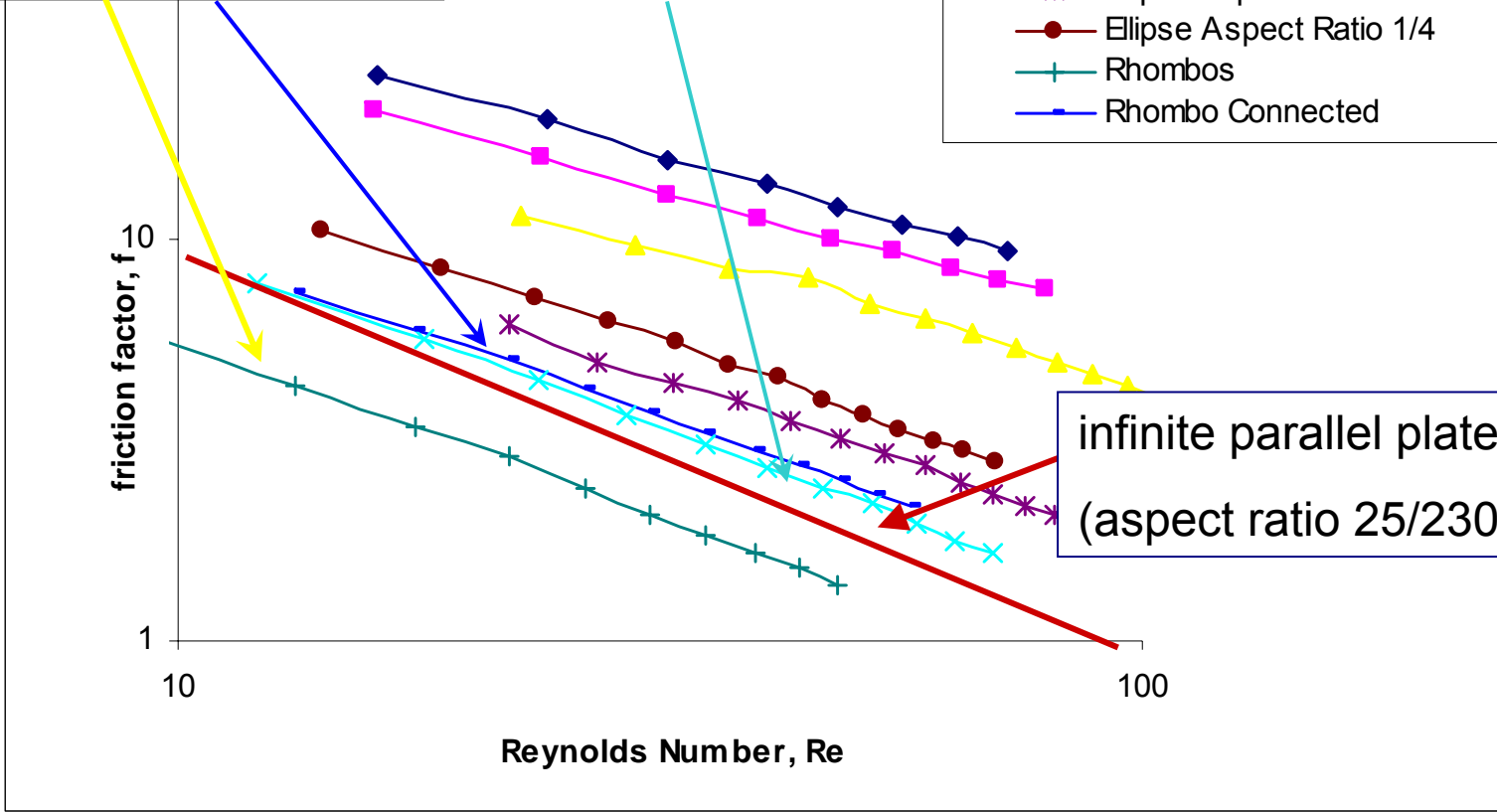
Pressure drop measurements





connections removed

- ◆ Circle Staggered
- Square Staggered
- ▲ Eye Shaped
- × Shifted Sine Connected
- * Ellipse aspect ratio 1/3
- Ellipse Aspect Ratio 1/4
- + Rhombos
- Rhombo Connected



Status - Outlook

- Micro cryocooler has high potential for space applications
- The total volume of the cold stage $\ll 1 \text{ cm}^3$, 10 mW @ 96 K
- Total integration of cold stage, device, vacuum chamber and compressor

Micro-recuperative cooling

- Optimal design of counter flow heat exchanger is complete,
- First prototypes are ready to be tested.
- Multi staging: 5 mW @ 27 K

Micro-regenerative cooling

- Various micro regenerator geometries were fabricated.
- Hydraulic and Thermal characterization of different geometries is done.

Acknowledgements



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